

# DESIGN AND STRUCTURAL ANALYSIS OF AUTOMATIC REVERSE LOCKING RACK MECHANISM FOR CAR JACK

Senthil Kishore V\*, S Periyasamy\*\*, N Nandakumar\*\*\*

\*(Engineering Design, Government College of Technology, Coimbatore, India  
Email: [sent.71772272109@gct.ac.in](mailto:sent.71772272109@gct.ac.in))

\*\*\*(Engineering Design, Government College of Technology, Coimbatore, India  
Email: [speriyasamy@gct.ac.in](mailto:speriyasamy@gct.ac.in))

\*\*\*\*(Engineering Design, Government College of Technology, Coimbatore, India  
Email: [nandakumar@gct.ac.in](mailto:nandakumar@gct.ac.in))

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## Abstract:

A screw jack is a type of jack which functions by the turning of a lead screw. It is commonly used to lift heavy load to a certain height. A good example is the car-jacks. In the case of a screw jack, due to the wear of trunnion's thread it may cause the slippage of the lead screw and leads to the sudden fall of the vehicle which is been lifted by the screw jack. The statement of the problem has led to the motivation of designing the automatic reverse locking rack mechanism for car jack. The automatic reverse locking rack mechanism is a proposed system and method for preventing a vehicle from sudden fall when jack fails to support the vehicle for certain period of time. The work presented here reveals the construction and design of the screw jack safety system. The design is made to be compact for the ease of use. The best suitable material for the components were taken into account which are to be analyzed by finite element analysis method. Along with the results that obtained can give the conclusion about the stress, strain and maximum load carrying capacity with minimum deformation.

**Keywords** —Screw jack, Automatic reverse locking rack mechanism, safety system.

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## I. INTRODUCTION

The jack is a device which is usually used to raise the significant masses. It is one of the most widely used accessories for lifting the vehicles. Whatever the problem would be like puncher of the tyre, jack is performed to lift the vehicle to certain height for a specific period of time. During this period in case of scissor jack, the power screw with certain friction coefficient helps for self-locking mechanism, however if the trunnion's internal thread is worn out. It may lead to sudden fall of the vehicle. To arrest the accidental fall. This automatic reverse locking rack mechanism plays its role to prevent the sudden fall of the vehicle. The rack type-ratchet pawl mechanism serves the purpose to allow the rack to move linearly in one direction only. It preventing the ratchet rack from falling in reverse direction.

Scissor jack is one of the powerful lifting jacks operated by a screw in a horizontal position. It gets lengthens and shortens the horizontal diagonal of the parallelogram which consisting of linkages of the jack. It consists of metal components that connects together in a scissor-like shape between the top and base of the jack. Those metal components act as an adjustable lift using the scissor mechanism by changing its height and to withstand a wide range of weight.

The rack type-ratchet pawl mechanism plays an important role in providing one way transmission and safety against uneven and heavy loading conditions. The ratchet is a rack that has teeth cut out of it and a pawl that follows as the rack moves longitudinally. In recent analysis of the fatal accident statistics which showed the reversing activities were involved in 12 percentage of all the fatal transport accidents. Accidents during reverse

movement results less in injury but more damage to vehicles and other human properties.

In this study, the safety system automatic reverse locking rack mechanism were designed and static structural strength were assessed.

## II. PROBLEM DEFINITION

The purpose of the study was defined below,

- Car jack has no safety system to arrest the fall.
- However, power screw with certain friction coefficient helps for self-locking mechanism.
- If the trunnion's internal thread is worn out. It may lead into sudden fall.
- To arrest that accidental fall and to give an additional support for the jack.
- This automatic reverse locking rack mechanism serves its purpose.

## III. METHODOLOGY

The objectives of the project are as follows,

- To develop structural modelling of the safety system with automatic reverse locking rack mechanism.
- To perform finite element analysis of the designed model.
- Suitable material study.
- To Study the load factors.
- To investigate the stress, deformation induced in the model.

### A. Materials

The aim of this project is to design and analyse the safety system by various parameters involved in it. This material of model will give more strength and will give long life. Rack type ratchet pawl mechanism is used here to provide one way motion only to arrest the sudden fall. The following are the materials that to be used in the components.

- EN9 (Normalize) for rack type ratchet component.
- C20 (Steel Forging) for pawl component.
- ASTM A36 STEEL (Mild Steel) for all the remaining components. The Table 1 shows the materials and their physical properties to be used in the components for analysis.

TABLE I  
MATERIAL AND THEIR PROPERTIES OF EXISTING AND SELECTED MATERIAL

Properties	Materials		
	ASTM A36 STEEL	C20	EN9
Density [Kg/m <sup>3</sup> ]	7850	7850	7800
Yield tensile strength [Mpa]	250	360	355
Ultimate tensile strength [Mpa]	550	425	700
Youngs modulus [Mpa]	200x10 <sup>3</sup>	210x10 <sup>3</sup>	206x10 <sup>3</sup>
Bulk modulus [Mpa]	140x10 <sup>3</sup>	175x10 <sup>3</sup>	171x10 <sup>3</sup>
Shear modulus [Mpa]	79x10 <sup>3</sup>	80x10 <sup>3</sup>	79x10 <sup>3</sup>
Poisson ratio	0.26	0.3	0.3

### B. Design calculations

The weight of the car is almost 1.5 ton, where the weight of the axle is about 60% of 1.5 ton. So, the weight of the front axle is considered to be 450 Kg. The axle lies on the rotatory support of the model which has the 80 mm diameter

$$\text{Weight} = 450 \text{ kg}$$

$$\text{Load} = 450 \times 9.81 = 4414.5 \text{ N}$$

$$\text{Diameter} = 80 \times 10^{-3} \text{ m}$$

$$\text{Area} = (3.141/4) \times (80 \times 10^{-3})^2 = 0.005026 \text{ m}^2$$

$$\text{Pressure} = 4414.5 / 0.005026 = 878332.67 \text{ N/m}^2$$

The pressure of 878332.67 pascal or N/m<sup>2</sup> that acts on the rotatory support. Then the model is to be analysed whether it can withstand that load which acts on it.

## IV. MODEL OF THE SAFETY SYSTEM

There are several components which are used for designing the safety system were modelled using SolidWorks whose dimensions and their respective views were shown below,

- 1) **Base plate:** It hold the system. The Fig. 1 shows the model of base plate with its dimensions.





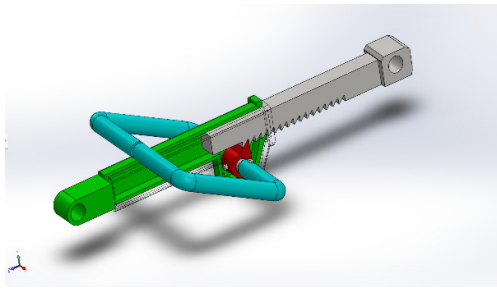


Fig. 11 Assembled view of automatic reverse locking rack mechanism

The Fig. 12 shows the assembled view of the safety system using automatic reverse locking rack mechanism for car jack. The foldable arms are used to fold the setup to make the system to be compact. The Fig. 13 shows the assembled view of folded safety system.

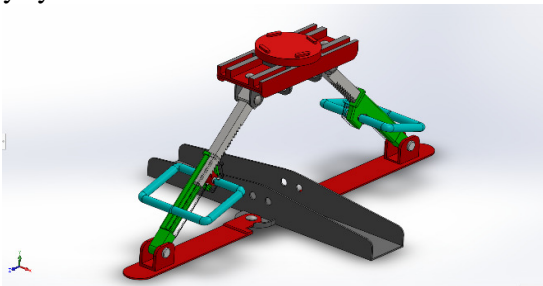


Fig. 12 Assembled view of safety system using automatic reverse locking rack mechanism for car jack

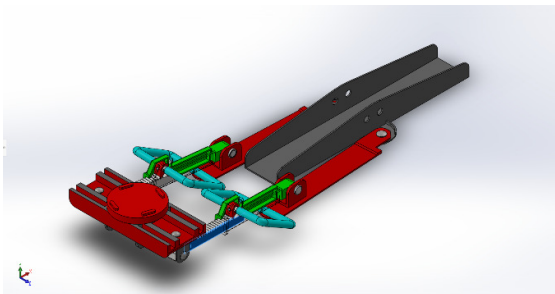


Fig. 13 Assembled view of folded safety system

## VI. FINITE ELEMENT ANALYSIS

**1. Analysis of safety system using automatic reverse locking rack mechanism for car jack:** The analysis of the safety system using automatic reverse locking rack mechanism model is carried out using ANSYS workbench software using finite element method. The analysis is done in the static structural method. Then the feed the material data. Import the geometry model in IGES format. Specify the materials for each component used in

the model. Mesh the model. Apply the boundary condition such as fixed support is assigned to the bottom plate; by assigning the fixed support to the bottom plate is constrained in all degree of freedom and it would withstand the load acting on it. Then apply 87332.670 pascal pressure which acts on the rotatory support. Solve the model for the solution of total deformation, equivalent stress(von-Mises), maximum principal stress, maximum shear stress, and equivalent elastic strain.

The figure 14, 15, 16, 17 and 18 shows the analysis for safety system using automatic reverse locking rack mechanism for car jack.

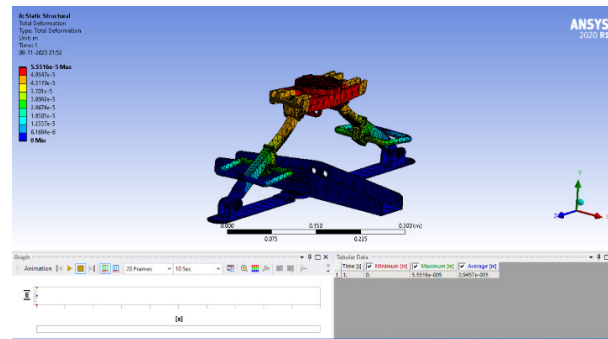


Fig. 14 Total deformation for safety system

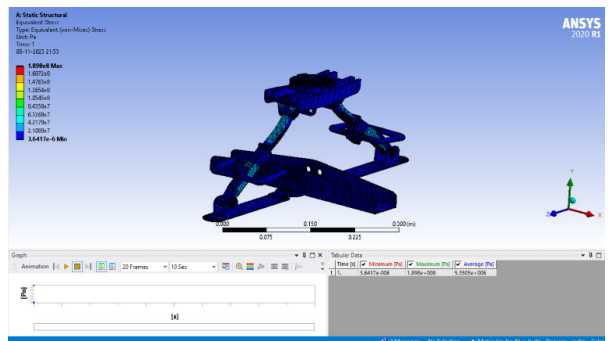


Fig. 15 Equivalent stress for safety system

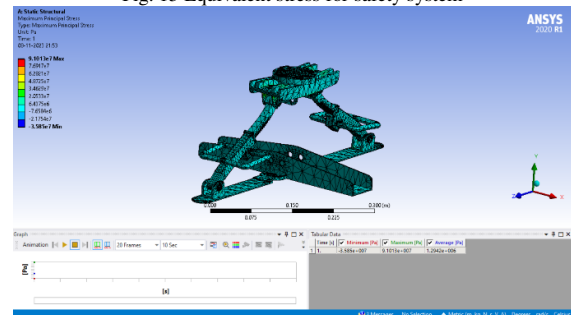


Fig. 16 Maximum principal stress for safety system



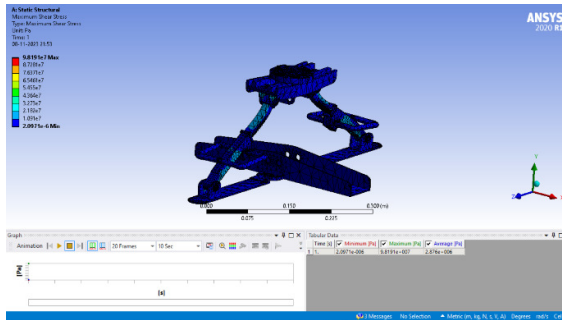


Fig. 17 Maximum shear stress for safety system

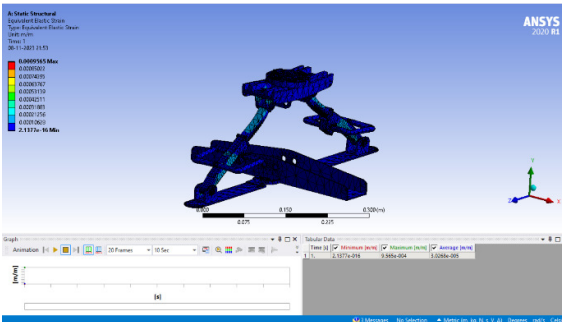


Fig. 18 Equivalent elastic strain for safety system

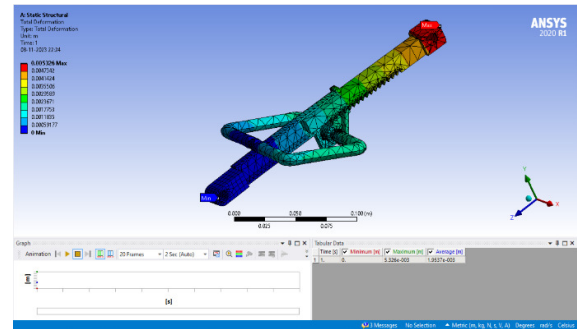


Fig. 19 Total deformation of automatic reverse locking rack mechanism

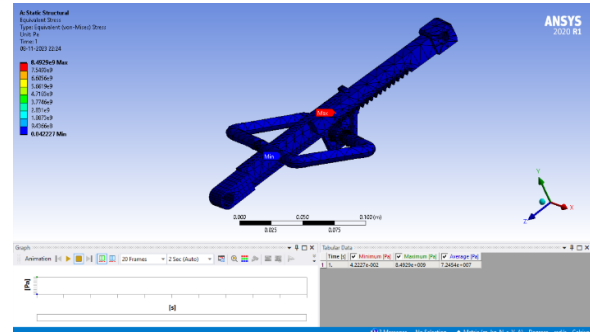


Fig. 20 Equivalent stress for automatic reverse locking rack mechanism

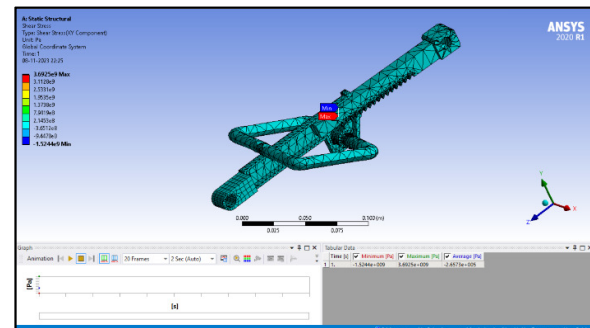


Fig. 21 Shear stress for automatic reverse locking rack mechanism

**2. Analysis of automatic reverse locking rack mechanism:** The analysis of the automatic reverse locking rack mechanism model is carried out using ANSYS workbench software using finite element method. The analysis is done in the static structural method. Then the material data can be obtained using material library or else give the corresponding property values manually. Import the geometry model in IGES format. Specify the materials for each component used in the model. Mesh the model. Apply the boundary condition such as fixed support is assigned to the bottom slot; by assigning the fixed support to the bottom slot is constrained in all degree of freedom and it would withstand the load acting on it. Using probe got the pressure which acts on the rack's head. Then apply  $1.1534 \times 10^7$  pascal pressure which acts on the rack type ratchets head. Solve the model for the solution of total deformation, equivalent stress(von-Mises), shear stress, etc. The Fig. 19, 20 and 21 shows the analysis for automatic reverse locking rack mechanism.

## VII. RESULTS AND DISCUSSION

It is observed that the total deformation, equivalent stress (von-mises stress), maximum principal stress, maximum shear stress and equivalent elastic strain for the safety system is obtained. The maximum stress is occurring at the top plate where it makes contact with the rack type ratchet.

The Table2 shows the results obtained for the safety system using automatic reverse locking rack mechanism for car jack.

Input parameter:

$$\text{Load} = 450 \times 9.81 = 4414.5 \text{ N}$$

$$\text{Area} = (3.141/4) \times (80 \times 10^{-3})^2 = 0.005026 \text{ m}^2$$

$$\text{Pressure} = 4414.5 / 0.005026 = 878332.67 \text{ N/m}^2$$

TABLE II  
RESULTS OBTAINED FOR SAFETY SYSTEM

	Range	Values
<b>Total deformation [m]</b>	<b>Maximum</b>	5.551e-5
	<b>Minimum</b>	0
<b>Equivalent stress [pa]</b>	<b>Maximum</b>	1.898e+8
	<b>Minimum</b>	3.641e-6
<b>Maximum principal stress [pa]</b>	<b>Maximum</b>	9.101e+7
	<b>Minimum</b>	-3.585e+7
<b>Maximum shear stress [pa]</b>	<b>Maximum</b>	9.819e+7
	<b>Minimum</b>	2.097e-6
<b>Equivalent elastic strain [m/m]</b>	<b>Maximum</b>	9.565e-4
	<b>Minimum</b>	2.137e-16

It is observed that the total deformation, equivalent stress (von-mises stress) and Shear stress for the automatic reverse locking rack mechanism is obtained. The Table3 shows the results obtained for automatic reverse locking rack mechanism. The maximum stress occurs at the contact point of rack type ratchet's tooth with the pawl.

TABLE III  
RESULTS OBTAINED FOR AUTOMATIC REVERSE LOCKING RACK MECHANISM

<b>Total deformation [m]</b>		<b>Equivalent stress [pa]</b>		<b>Shear stress [pa]</b>	
<b>Max</b>	<b>Min</b>	<b>Max</b>	<b>Min</b>	<b>Max</b>	<b>Min</b>
3.463 e-10	0	552.25	2.304 e-9	240.1	-99.126

## VIII. CONCLUSION

From finite element analysis of the safety system using automatic reverse locking rack mechanism for car jack the following conclusions can be drawn.

- Total deformation of the model is less under loading condition.

• Overall, the model of the safety system using automatic reverse locking rack mechanism for car jack gives nearly maximum stress with minimum deformation value in static structural analysis.

• Hence it can withstand the vehicle load even when the car jack fails its function.

• Also, it can be used for the supporting system of the car jack.

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